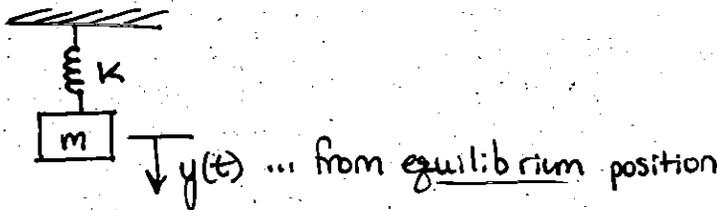


Differential Equations in Mechanical Systems

Differential Equations in Mechanical Systems: (ME2210, BME3212, ME4140, ME4210)

EXAMPLE: Free vibration of a Spring-Mass system:



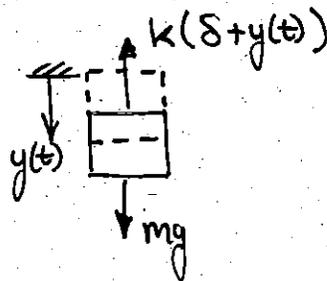
(i) Free-body Diagram @ equilibrium ( $y=0$ )



$\delta$  = equilibrium elongation of spring

$$\sum F_y = 0 : k\delta = mg \Rightarrow \delta = \frac{mg}{k}$$

(ii) Free-body Diagram @  $y > 0$  (mass is now in motion!)



$$\sum F_y = ma = m\ddot{y}$$

$$mg - k(\delta + y(t)) = m\ddot{y}$$

$$mg - k\delta - ky(t) = m\ddot{y}$$

but  $\delta = \frac{mg}{k}$

$$\Rightarrow mg - k\left(\frac{mg}{k}\right) - ky(t) = m\ddot{y}$$

$$\boxed{m\ddot{y} + ky(t) = 0} \quad + \text{Initial Conditions}$$

(second-order differential equation)

note - second derivative  $\Rightarrow$  second order!

General Solution:(1) Transient (homogeneous/complementary) Solution:

Assume  $y = e^{st}$

$$\Rightarrow \dot{y} = se^{st} \quad ; \quad \ddot{y} = s^2 e^{st}$$

plug into  $m\ddot{y} + ky = 0$

$$\Rightarrow m(s^2 e^{st}) + k(e^{st}) = 0$$

$$e^{st}(ms^2 + k) = 0$$

$$\therefore s^2 = -\frac{k}{m} \Rightarrow s = \pm\sqrt{-\frac{k}{m}} = \pm j\sqrt{\frac{k}{m}} \quad (j = \sqrt{-1})$$

two roots:  $s_1 = j\sqrt{\frac{k}{m}} \quad ; \quad s_2 = -j\sqrt{\frac{k}{m}}$

$$\therefore y_{\text{trans}}(t) = C_1 e^{s_1 t} + C_2 e^{s_2 t}$$

$$\therefore y_{\text{trans}}(t) = C_1 e^{j\sqrt{\frac{k}{m}}t} + C_2 e^{-j\sqrt{\frac{k}{m}}t}$$

(2) Steady-State: since RHS = 0,  $y_{\text{ss}}(t) = 0$   
(no forcing function)

(3) General Solution:  $y(t) = y_{\text{trans}}(t) + y_{\text{ss}}(t)$   
$$\Rightarrow y(t) = C_1 e^{j\sqrt{\frac{k}{m}}t} + C_2 e^{-j\sqrt{\frac{k}{m}}t}$$

Recall from complex #'s,

$$e^{j\theta} = \cos\theta + j\sin\theta$$

(Euler's Formula)

↑ pronounced "Oiler's"!

we can simplify general solution above,

$$\Rightarrow y(t) = C_1 e^{j\sqrt{\frac{k}{m}}t} + C_2 e^{-j\sqrt{\frac{k}{m}}t}$$

$$\cos(\sqrt{\frac{k}{m}}t) + j\sin(\sqrt{\frac{k}{m}}t)$$

$$\cos(-\sqrt{\frac{k}{m}}t) + j\sin(-\sqrt{\frac{k}{m}}t)$$

$$\begin{aligned} \Rightarrow y(t) &= C_1 [\cos(\sqrt{\frac{k}{m}}t) + j\sin(\sqrt{\frac{k}{m}}t)] + C_2 [\cos(-\sqrt{\frac{k}{m}}t) + j\sin(-\sqrt{\frac{k}{m}}t)] \\ &= C_1 [\cos(\sqrt{\frac{k}{m}}t) + j\sin(\sqrt{\frac{k}{m}}t)] + C_2 [\cos(\sqrt{\frac{k}{m}}t) - j\sin(\sqrt{\frac{k}{m}}t)] \\ &= \underbrace{(C_1 + C_2)}_{C_3} \cos(\sqrt{\frac{k}{m}}t) + \underbrace{(C_1 - C_2)j}_{C_4} \sin(\sqrt{\frac{k}{m}}t) \end{aligned}$$

NOTE:  $C_1$  &  $C_2$  must be complex conjugates for  $y(t)$  to be real

$$\therefore y(t) = C_3 \cos(\sqrt{\frac{k}{m}}t) + C_4 \sin(\sqrt{\frac{k}{m}}t)$$

(3) Initial Conditions: eg. initial displacement & velocity

$$\Rightarrow y(0) = A, \quad \dot{y}(0) = 0 \quad (\text{require 2 initial conditions for 2<sup>nd</sup> order D.E.})$$

$$y(0) = C_3 \overset{(\text{1})}{\cos(0)} + C_4 \overset{(\text{1})}{\sin(0)} = A \quad \rightarrow \therefore C_3 = A$$

$$\Rightarrow y(t) = A \cos(\sqrt{\frac{k}{m}}t) + C_4 \sin(\sqrt{\frac{k}{m}}t)$$

$$\dot{y}(t) = -A \sqrt{\frac{k}{m}} \sin(\sqrt{\frac{k}{m}}t) + C_4 \sqrt{\frac{k}{m}} \cos(\sqrt{\frac{k}{m}}t)$$

$$\dot{y}(0) = -A \sqrt{\frac{k}{m}} \overset{(\text{1})}{\sin(0)} + C_4 \sqrt{\frac{k}{m}} \overset{(\text{1})}{\cos(0)} = 0 \quad \rightarrow \therefore C_4 = 0$$

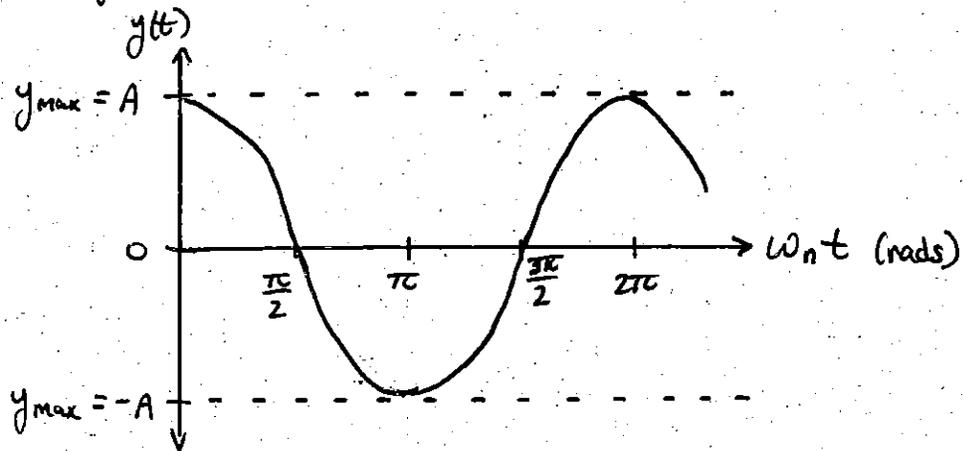
Total Solution:  $y(t) = A \cos(\sqrt{\frac{k}{m}}t)$  OR  $y(t) = A \cos(\omega_n t)$

$\omega_n = \sqrt{k/m}$  rad/s ... natural frequency of system

↪ increases with stiffness ( $k \uparrow \Rightarrow \omega_n \uparrow$ )

decreases with mass ( $m \uparrow \Rightarrow \omega_n \downarrow$ )

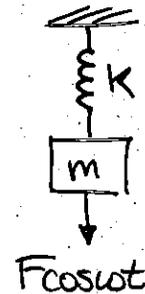
(general behavior of free vibrating system)



$\Rightarrow$  What if the mass is subjected to an applied force  $f(t)$ ?  
 (e.g.  $f(t) = F \cos \omega t \equiv$  forced vibration)

Governing Equation:

$$m\ddot{y} + ky(t) = F \cos \omega t$$



(1) Transient Solution: (unchanged!)

$$y_{\text{trans}}(t) = C_3 \cos \sqrt{\frac{k}{m}} t + C_4 \sin \sqrt{\frac{k}{m}} t$$

(2) Steady-state solution:  $m\ddot{y} + ky = F \cos \omega t$

Recall that the steady-state solution has the same form as the forcing function!

Since the forcing function  $f(t)$  is  $f(t) = F \cos \omega t$  (sinusoidal)

we assume  $y_{\text{ss}}(t) = A \sin \omega t + B \cos \omega t$  (also sinusoidal)

$$\Rightarrow \dot{y}_{\text{ss}}(t) = A \omega \cos \omega t - B \omega \sin \omega t$$

$$\Rightarrow \ddot{y}_{\text{ss}}(t) = -A \omega^2 \sin \omega t - B \omega^2 \cos \omega t$$

Plugging back into differential equation:

$$m(-A \omega^2 \sin \omega t - B \omega^2 \cos \omega t) + k(A \sin \omega t + B \cos \omega t) = F \cos \omega t$$

We need to solve now for constants  $A$  &  $B$ , grouping terms,

$$(Ak - Am\omega^2)\sin\omega t + (Bk - Bm\omega^2)\cos\omega t = F\cos\omega t$$

Comparing coefficients on  $\sin\omega t$  &  $\cos\omega t$  on both sides,

$$\sin\omega t: Ak - Am\omega^2 = 0 \quad \therefore A = 0$$

$$\cos\omega t: Bk - Bm\omega^2 = F \quad \therefore B = \frac{F}{k - m\omega^2}$$

$$\therefore y_{ss} = A\sin\omega t + B\cos\omega t = 0 + \left(\frac{F}{k - m\omega^2}\right)\cos\omega t$$

$$\therefore y_{ss} = \frac{F}{k - m\omega^2} \cos\omega t$$

(3) Total Solution:  $y(t) = y_{trans}(t) + y_{ss}(t)$

$$y(t) = C_3 \cos\left(\sqrt{\frac{k}{m}}t\right) + C_4 \sin\left(\sqrt{\frac{k}{m}}t\right) + \left(\frac{F}{k - m\omega^2}\right)\cos\omega t$$

(4) Initial Conditions:  $y(0) = 0$  &  $\dot{y}(0) = 0$

$$y(0) = C_3 \cos(0) + C_4 \sin(0) + \left(\frac{F}{k - m\omega^2}\right)\cos(0) = 0$$

$$C_3 + \left(\frac{F}{k - m\omega^2}\right) = 0 \quad \therefore C_3 = \frac{-F}{k - m\omega^2}$$

$$\dot{y}(0) = -C_3 \sqrt{\frac{k}{m}} \sin(0) + C_4 \sqrt{\frac{k}{m}} \cos(0) + \left(\frac{F}{k - m\omega^2}\right)\omega \sin(0) = 0$$

$$\therefore C_4 = 0$$

$$\therefore y(t) = \frac{F}{k - m\omega^2} (\cos\omega t - \cos(\sqrt{\frac{k}{m}}t))$$

What happens to  $y(t)$  as  $\omega \rightarrow \sqrt{\frac{k}{m}}$ ?

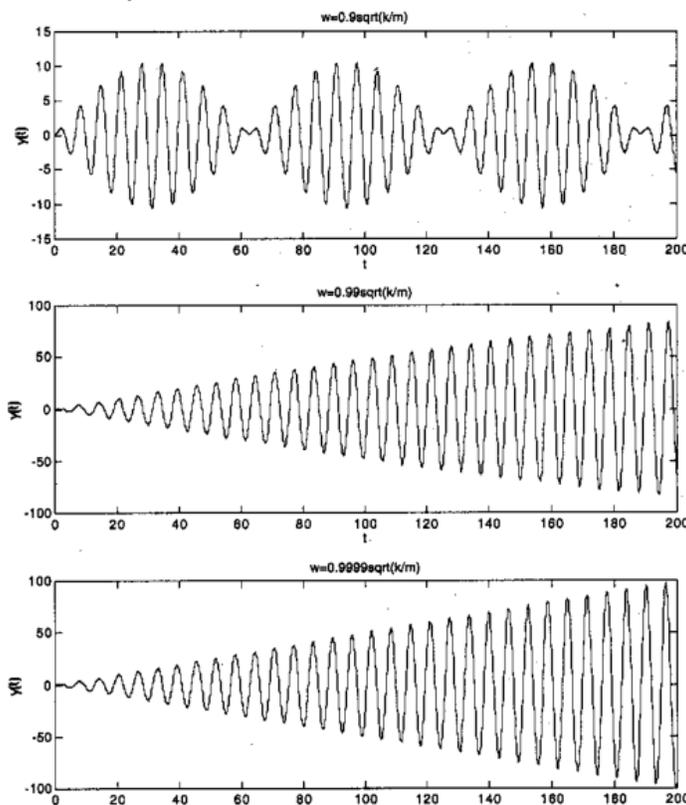
as  $\omega \rightarrow \sqrt{\frac{k}{m}}$ ,

$$y(t) = \left(\frac{F}{0}\right) \left(\cos\left(\sqrt{\frac{k}{m}}t\right) - \cos\left(\sqrt{\frac{k}{m}}t\right)\right) = \frac{0}{0}$$

This is an "indeterminate" form, and can be evaluated by methods of calculus not yet available to all students.

Instead, the results can be investigated by picking values of  $\omega$  close to  $\sqrt{k/m}$  and plotting the results,

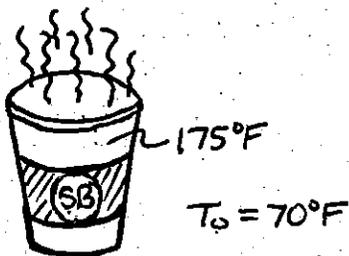
(e.g. let  $k=m=F=1$  and choose values of  $\omega = 0.9\sqrt{\frac{k}{m}}$ ,  $0.99\sqrt{\frac{k}{m}}$ , and  $0.9999\sqrt{\frac{k}{m}}$ )



The first plot shows the "beating" phenomenon typical of problems where forcing frequency  $\omega$  is in the neighborhood of natural freq.  $\omega_n = \sqrt{k/m}$ . As  $\omega$  is increased to  $0.99\sqrt{\frac{k}{m}}$  and  $0.9999\sqrt{\frac{k}{m}}$ , the last two plots show  $y(t)$  increasing without bound. This is called resonance, and is generally undesirable!

EXAMPLE: Newton's Law of Cooling (1<sup>st</sup> ORDER D.E.)

Consider a hot coffee initially at  $T(0) = 175^\circ\text{F}$  sitting in a room temperature environment @  $T_\infty = 70^\circ\text{F}$



The temperature  $T(t)$  of the coffee can be approx. by Newton's Law of Cooling as:

$$\frac{dT}{dt} + kT(t) = kT_\infty$$

$T(t)$  ... temperature of coffee ( $^\circ\text{F}$ )

$T_\infty$  ... temp. of ambient air ( $^\circ\text{F}$ )

$k$  ... convective heat transfer coefficient ( $\frac{\text{Btu}}{\text{ft}^2 \cdot \text{h} \cdot \text{F}}$ )

Find: Temperature of coffee,  $T(t)$

(1) Transient Solution:  $\text{RHS} = 0$ ,  $\dot{T}(t) + kT = 0$

assume,  $T_{\text{trans}}(t) = e^{st}$ ,  $\dot{T}_{\text{trans}}(t) = se^{st}$

$$se^{st} + ke^{st} = 0$$

$$e^{st}(s + k) = 0 \quad \therefore s = -k$$

$$\therefore T_{\text{trans}}(t) = C_1 e^{-kt}$$

(2) Steady-State Solution:  $\dot{T}(t) + kT = kT_\infty$  where  $T_\infty = 70^\circ\text{F}$

assume  $T_{\text{ss}}(t) = A$  (constant) since  $kT_\infty = \text{constant}$

$$\dot{T}_{\text{ss}}(t) = 0$$

$$0 + kA = kT_\infty \quad \therefore A = T_\infty = 70^\circ\text{F}$$

$$\therefore T_{\text{ss}}(t) = 70^\circ\text{F}$$

(3) Total Solution:  $T(t) = T_{\text{trans}}(t) + T_{\text{ss}}(t)$

$$T(t) = C_1 e^{-kt} + 70 \text{ } ^\circ\text{F}$$

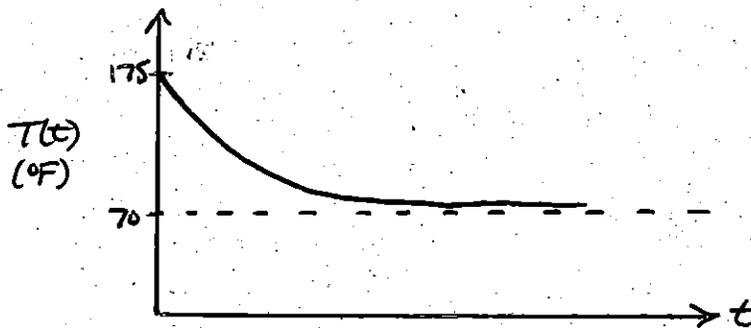
(4) Initial Conditions:  $T(0) = 175 \text{ } ^\circ\text{F}$

$$T(0) = C_1 e^0 + 70 = 175 \quad \therefore C_1 = 105 \text{ } ^\circ\text{F}$$

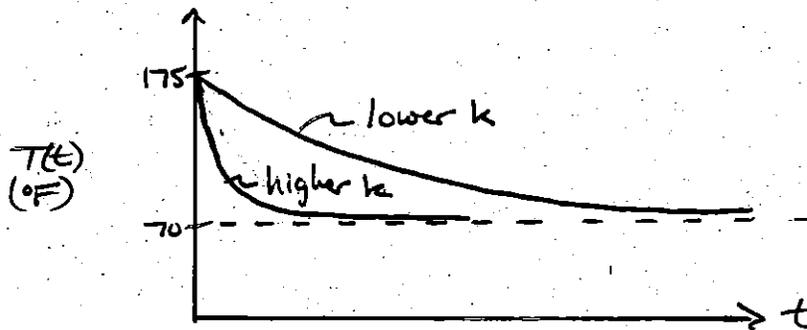
$$\therefore T(t) = 105 e^{-kt} + 70 \text{ } ^\circ\text{F}$$

$$t = 0, T = 175 \text{ } ^\circ\text{F}$$

$$t \rightarrow \infty, T \rightarrow 70 \text{ } ^\circ\text{F} \text{ (steady-state)}$$



What do changes in  $k$  (heat trans. coefficient) do to  $T(t)$ ?



Is lower  $k$  or higher  $k$  best for a cup of coffee?